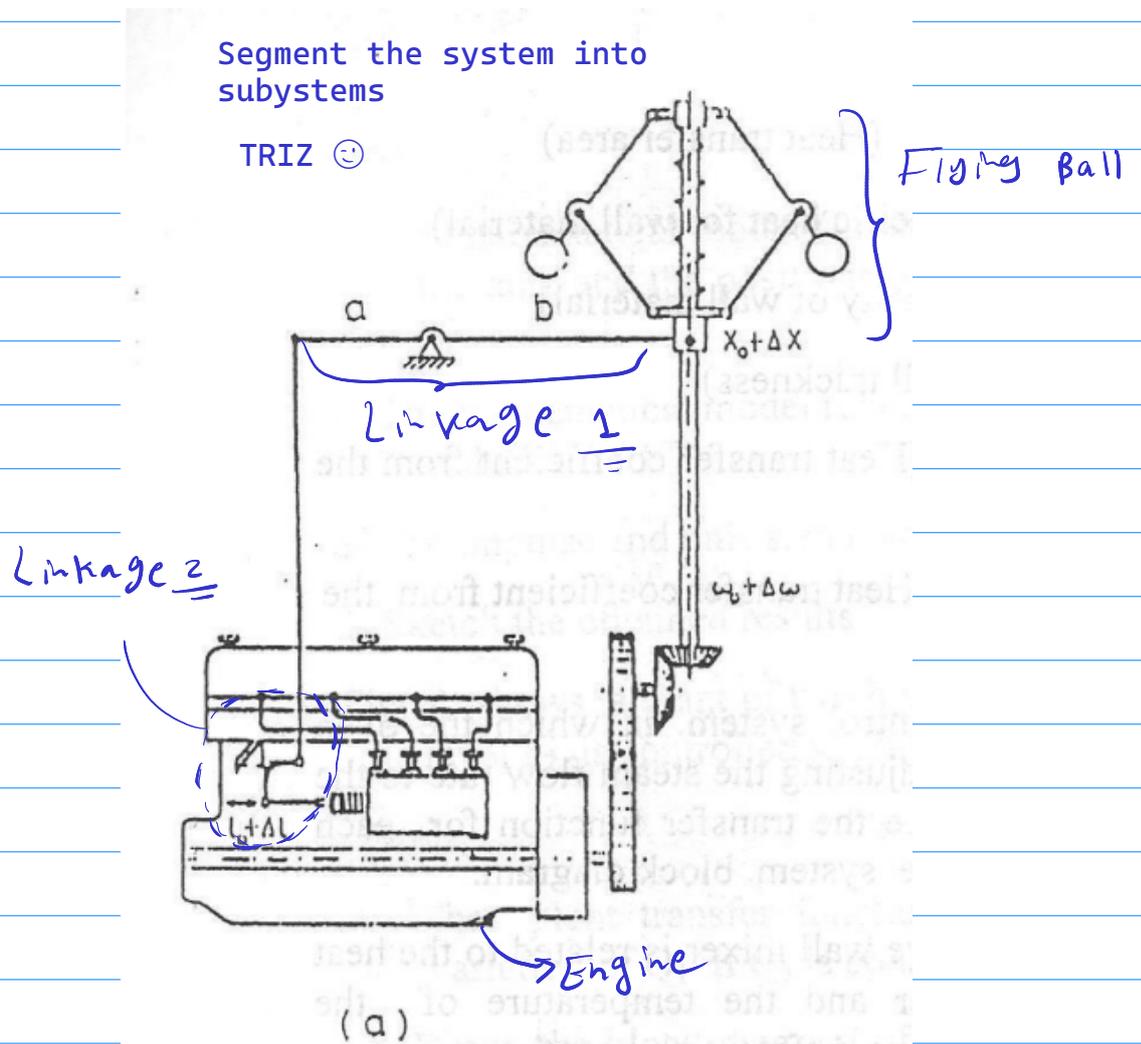
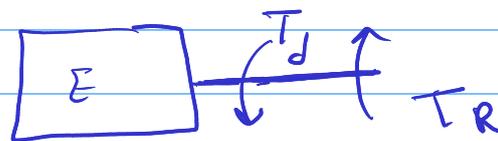


Derive the mathematical models of each component in the systems shown in fig (1) then construct the equivalent closed loop system on each case.



* Engine



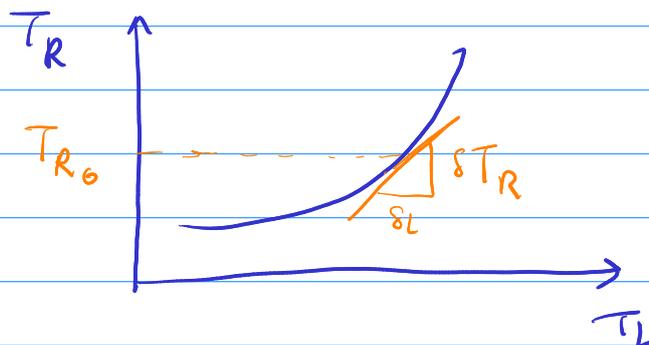
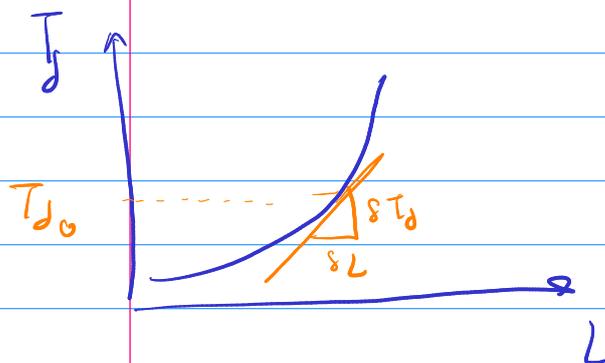
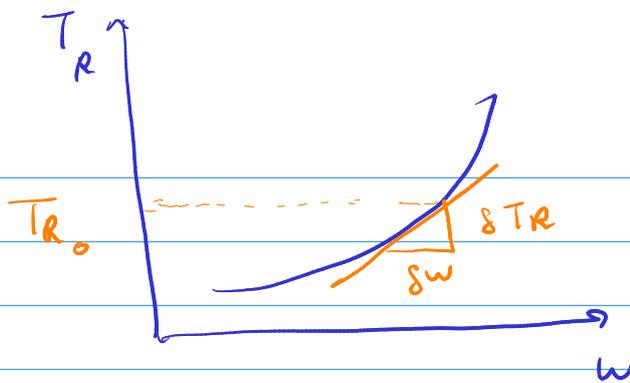
$$T_{net} = J_{ev} \dot{\omega} = T_d = T_R$$

$$T_d = f(\omega, l)$$

$$T_R = g(\omega, T_L)$$

Developed torque is a function of angular speed and throttled rack length

Resistive torque is a function of angular speed and load torque



f and g are non-linear functions

use taylor expansion
for linearization

$$T_d \approx T_{d0} + \frac{\partial T_d}{\partial w} \delta w + \frac{\partial T_d}{\partial L} \delta L$$

$$T_R \approx T_{R0} + \frac{\partial T_R}{\partial w} \delta w + \frac{\partial T_R}{\partial L} \delta L$$

you can read the above two equations as follow:

- developed torque is equal to developed torque at equilibrium + tiny changes in developed torque caused by tiny changes in angular speed and tiny changes in throttle rack length

- resistive torque is equal to resistive torque at equilibrium + tiny changes in resistive torque caused by tiny changes in angular speed

since we are linearizing T_d and T_R , then the derivatives are const:

$$\frac{\partial T_d}{\partial L} = c_1$$

$$\frac{\partial T_d}{\partial w} = c_2$$

$$\frac{\partial T_R}{\partial L} = c_3$$

$$\frac{\partial T_R}{\partial w} = c_4$$

$$T_d = T_{d_0} + c_1 \Delta L + c_2 \Delta w$$

$$T_R = T_{R_0} + c_3 \Delta T_L + c_4 \Delta w$$

at equilibrium: $T_d = T_R \rightarrow T_{d_0} = T_{R_0}$

$$T_{net} = J_{\text{env}} \dot{w} = T_d - T_R = c_1 \Delta L + (c_2 - c_4) \Delta w - c_3 \Delta T_L$$

$$T(s) = J_{\text{env}} s w = c_1 L(s) + [c_2 - c_4] w(s) - c_3 T_L(s)$$

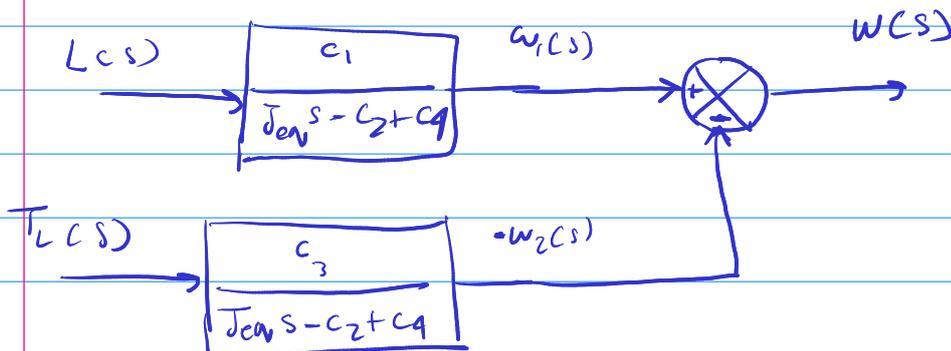
$$[J_{\text{env}} s - c_2 + c_4] w(s) = c_1 L(s) - c_3 T_L(s)$$

to get a transfer function for $w(s)$, we need to use superposition principle

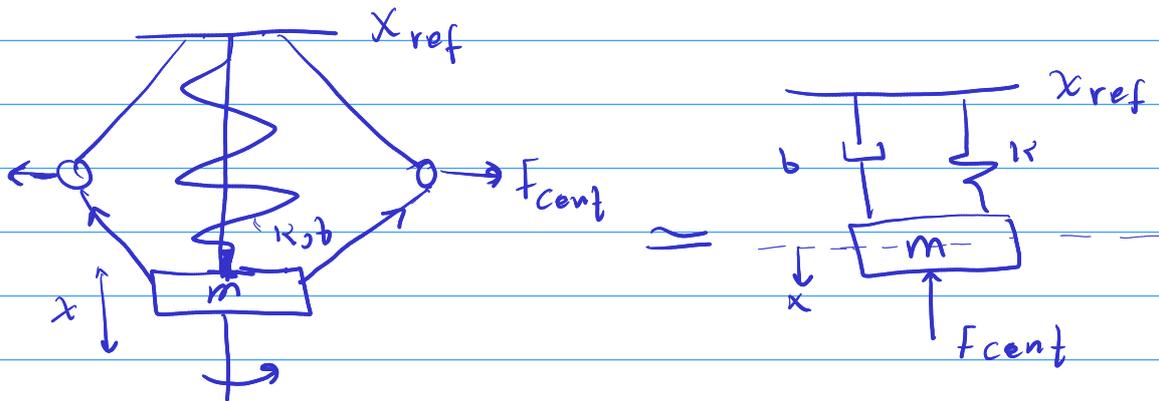
$$\text{@ } T_L(s) = 0 \rightarrow \frac{w_1(s)}{L(s)} = \frac{c_1}{J_{\text{env}} s - c_2 + c_4}$$

$$\text{@ } L(s) = 0 \rightarrow \frac{w_2(s)}{T_L(s)} = \frac{-c_3}{J_{\text{env}} s - c_2 + c_4}$$

to get $w(s)$, we need to sum $w_1(s)$ (the influence of $L(s)$ on $w(s)$) and $w_2(s)$ (the influence of $T_L(s)$ on $w(s)$)



* Flying Ball



Note:
 $\dot{x}_{ref} = 0$
 $\ddot{x}_{ref} = 0$

$$F_{net} = m \ddot{x} = F_{cent} - k(x - x_{ref}) - b \dot{x}$$

F_{cent} , centripital force, is a function of w

let:

$$F_{cent} \propto w$$

$$F_{cent} = C_{cent} w$$

$$\mathcal{L} \left(m \ddot{x} = C_{cent} w - k(x - x_{ref}) - b \dot{x} \right)$$

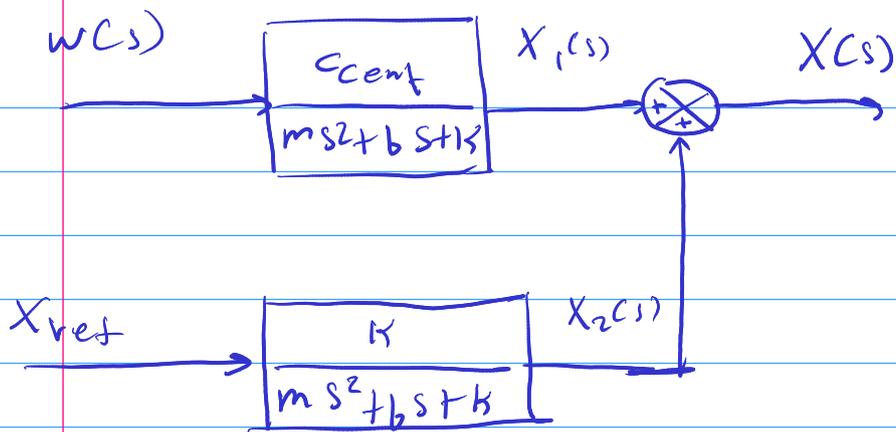
$$\left[m s^2 + b s + k \right] X(s) = k X_{ref}(s) + C_{cent} W(s)$$

to get a transfer function for $x(s)$, we need to use superposition

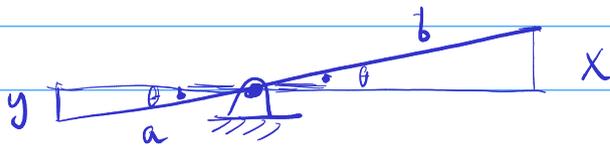
$$\textcircled{a} X_{ref}(s) = 0 \rightarrow \frac{X_1(s)}{W(s)} = \frac{C_{cent}}{m s^2 + b s + k}$$

$$\textcircled{a} W(s) = 0 \rightarrow \frac{X_2(s)}{X_{ref}(s)} = \frac{k}{m s^2 + b s + k}$$

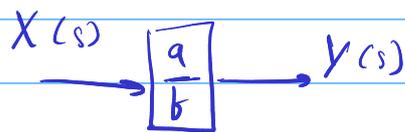
to get $X(s)$, we need to sum $X_1(s)$ (the influence of $w(s)$ on $X(s)$) and $X_2(s)$ (the influence of $X_{ref}(s)$ on $X(s)$)



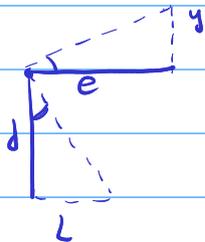
* Linkage 1



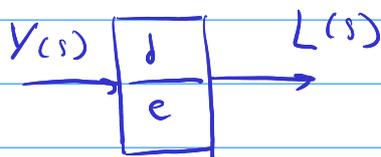
$$\sin \theta = \frac{y}{a} = \frac{x}{b} \xrightarrow{L} Y(s) = \frac{a}{b} X(s)$$



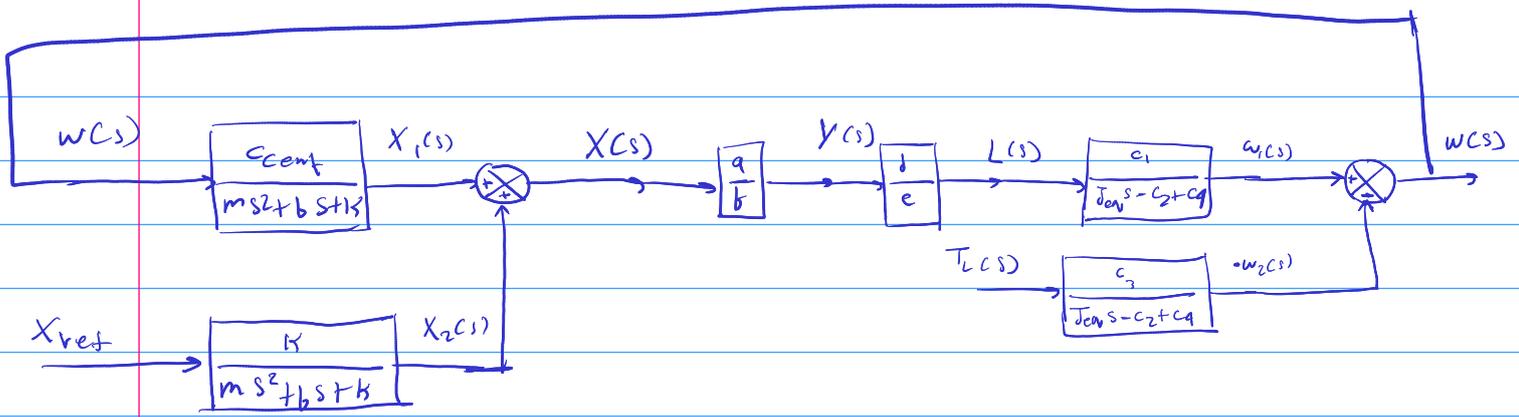
* Linkage 2

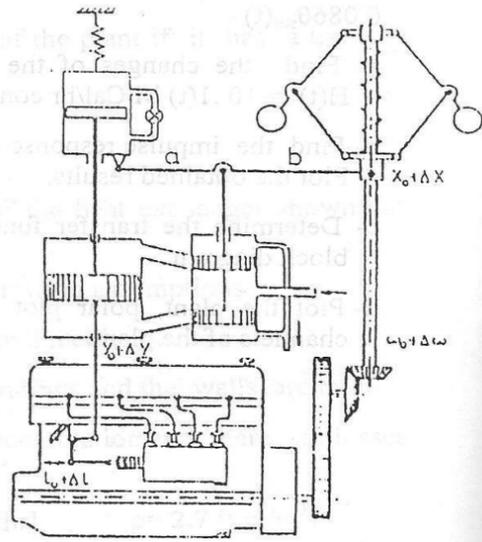
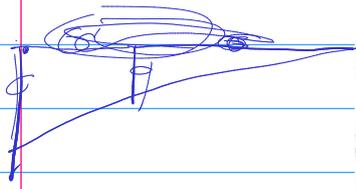


$$\tan \theta = \frac{y}{e} = \frac{x}{d} \xrightarrow{L} L(s) = \frac{d}{e} Y(s)$$



System





(d)

$$J\ddot{w} = T_{act} - T_R$$

$$T_D = f(L, w) \quad T_R = g(T_L, w)$$

* Engine: $Jsw(s) = c_1 L(s) + c_2 w(s) - c_3 T_L(s) + c_4 w(s)$

$$w(s) = \frac{c_1}{Js + c_4 - c_2} L(s) - \frac{c_3}{Js + c_4 - c_2} T_L(s)$$

cent
miser ← f(x) = F(w) = F(x_ref) → k

* Flyball: $[ms^2 + bs + k]X(s) = c_{cent} W(s) - k X_{ref}(s)$

$$X(s) = \frac{c_{cent}}{ms^2 + bs + k} W(s) = \frac{k}{ms^2 + bs + k} X_{ref}(s)$$

* Linkage: $Y(s) = \frac{a}{b+a} X(s) - \frac{b}{b+a} Z(s)$

b ← f(x) f(x) → m, b, k

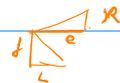
* spool valve: $b_2 s R(s) = [ms^2 + b_2 s + k_2] Z(s)$

$$R(s) = \frac{ms^2 + b_2 s + k_2}{b_2 s} Z(s)$$

ch h(?) → s

* Tank: $ASR(s) = c_5 Y(s)$

$$R(s) = \frac{c_5}{As} Y(s)$$



* Linkage 2: $R(s) = \frac{e}{J} L(s)$

* Variables:

$w(s), X(s), Y(s), Z(s), R(s), L(s)$

* Engine:

$$w(s) = \frac{c_2}{Js + c_5} L(s) - \frac{c_3}{Js + c_5} T_{ext}(s)$$

* flyball sensor:

$$X(s) = \frac{K}{ms^2 + bs + K} X_{ref}(s) + \frac{c_{cont}}{ms^2 + bs + K} w(s)$$

* Lever:

$$Y(s) = \frac{a}{a+b} X(s) - \frac{b}{a+b} Z(s)$$

* Spool valve:

$$q_{in} = q_o + q_{stand} \Rightarrow q_{in} = A \frac{dR}{dt} = c_y \dot{y}$$
$$A s R(s) = c_y Y(s)$$

$$R(s) = \frac{c_y}{A s} Y(s)$$

&

$$Z(s) = \frac{b_2 s}{m_{asy} s^2 + b_2 s + k_2} R(s)$$

&

$$L(s) = \frac{d}{c} R(s)$$

